

# Silica Research in Glasgow

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**Abstract.** The Glasgow group is involved in the construction of the GEO600 interferometer as well as in R&D activity on technology for advanced gravitational wave detectors. GEO600 will be the first GW detector using quasi-monolithic silica suspensions in order to decrease thermal noise significantly with respect to steel wire suspensions. Results concerning GEO600 suspension mounting and performance will be shown in the first section. Section 2 is devoted to the present results from the direct measurement of thermal noise in mirrors mounted in the 10m interferometer in Glasgow which has a sensitivity limit of  $4 \cdot 10^{-19}$  m/sqrt(Hz) above 1kHz. Section 3 presents results on the measurements of coating losses. R&D activity has been carried out to understand better how thermal noise in the suspensions affects the detector sensitivity, and in Section 4 a discussion on Non Linear Thermoelastic effect is presented.

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## 1. Silica Suspensions in GEO600

The interferometer GEO600 is designed to detect gravitational waves above 50 Hz. At such frequencies the internal thermal noise in the mirrors is expected to give the main limitation to the detector sensitivity. The other two relevant mechanical noises, seismic noise and suspension thermal noise, do not give a significant contribution due to the suspension isolation system and the relatively low frequencies and low mechanical losses of the suspension resonances. Creep in suspension fibres may introduce a significant continuous noise or generate spurious bursts in the detector. Measures have been made on steel wires [1] for creep rate and events detection showing that the contribution of creep for steel suspension is not relevant. From the same authors a measure on creep rate for a loaded silica fibre has been reported on [2]. Due to the low rate measured, continuous noise coming from creep should be well below the thermal noise level. A group in the Moscow State University is working on the direct measurement of creep events in silica fibres in order to tackle the problem of spurious bursts [3].

Energy loss mechanisms are sources of thermal noise. Due to contact friction, mirror suspension systems realized with steel wires give a major contribution to the energy loss in the internal dynamics of test masses. In order to reduce this energy loss and then reduce the thermal noise in the detection bandwidth, GEO600 uses a special bonding technique (silicate bonding) to attach mirrors to the suspensions systems. In this technique small silica prisms are chemically bonded on flat surfaces machined on the barrel of each mirror. Silica fibres are then welded on these silica prisms realizing what is called a quasi monolithic suspension. In this system all internal frictional mechanisms are avoided and the high quality factor of silica is then preserved.

Fibres have been pulled using a computer driven machine. Silica rods (Suprasil 3) of 8 cm length, 5 mm diameter, are melted in a middle region using an oxy-hydrogen flame coming out from 5 nozzles. Then, they are pulled for approximately 2 cm in order to create a section having a diameter of about 2 mm. Finally the fibre is pulled to the length of 285 mm once this thin section is melted again. The first pulling makes the fibre neck sufficiently thin to allow easy cutting later on. Several fibres were produced in the AEI Intitute in Hannover and then transported to the GEO600 site in a special box.

The pulling machine used was electronically controlled in order to produce a set of fibres as homogeneous as possible in terms of diameter and length. Differences of the order of few mm in length are recovered through the cutting process. So, the only relevant fibre parameter is the longitudinal spring constant which has been measured through the bouncing mode resonant frequency. In this measurement each fibre was suspended with a load of 1.4 kg that is exactly the nominal load expected in GEO600. Fibres used had resonant frequencies lying between 16 and 16.5 Hz.

Due to the high quality silica used and the high dilution factor, the violin modes  $Q_s$  are foreseen to be of the order of  $10^8$ . This corresponds to a characteristic decay time of about 2 days for the first mode (500 Hz). Unexpected excitations of violin modes could compromise the locking performances for too long a time. In order to reduce this time all the fibres used in this first suspension were coated with a small amount of Teflon on two sections 5 mm long: one in the middle and the other at a third of the total fibre length. In this way a non homogeneous loss was created. The particular position of the coated sections was chosen to damp particularly the first modes while affecting the dynamics at frequencies below hundred Hertz in a negligible way. Teflon was applied through the deposition of drops of a solution of Teflon AP in a solvent called Flourinert.

Fibre strength was tested at 5 kg after coating. After this, cutting was initiated by scratching the necks with a diamond file. The correct position of the scratches was fixed using a jig where the fibres were held. Length reproducibility is of the order of 0.5 mm.

Each fibre was welded to the tip of a prism using the small neck left from the cutting. Again an oxy-hydrogen flame was used to melt the silica. Several types of hypodermic needles were used as nozzles. The upper neck was welded at first, holding the fibre with steel tweezers as close to the neck as possible. Any resulting scratches have been cured by melting the silica where the neck was held by the tweezers. Discrepancies between the fibre length and the distance between prism tips were fixed using melted silica from thin rods or by adjusting the vertical position of one neck using tweezers again.

## 2. Direct measurement of thermal noise

In the past year the Glasgow 10m prototype interferometer with Fabry-Perot cavities in the arms has been upgraded using a high power IR laser (800 mW). As a result of this work, the sensitivity limit was pushed down below  $10^{-18}$  m/sqrt(Hz) above 600 Hz, as shown in spectrum a) in Figure 3. Several noise sources contributed to this limit and it was not possible to estimate them separately. In particular was not clear how much displacement noise came from the thermal noise associated with the internal modes of the suspended optics.

In order to measure the internal thermal noise, it was decided to increase the internal damping of one of the mirrors using three strips of vacuum grease placed along the barrel, and then spectrum b) in Figure 3 was recorded. Time lag between the recording of spectrum a) and the spectrum b) was kept as short as possible compatible with the re-making of vacuum in the tank. Clearly an increase of the noise floor was seen suggesting that the increase may come from the thermal noise of the greased mass.

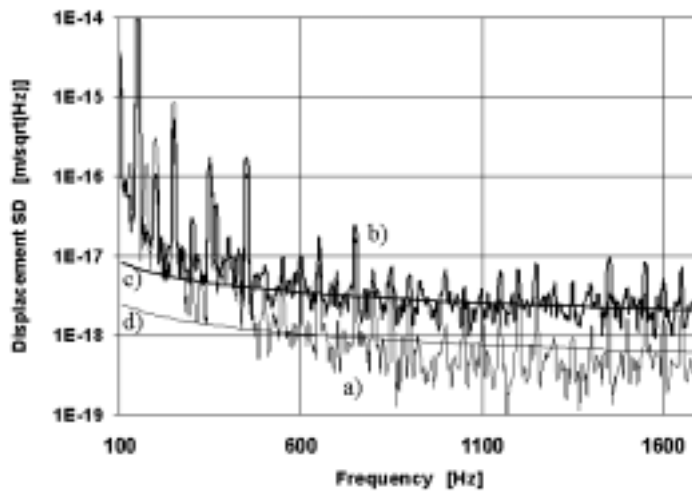


**Figure 1.** Intermediate mass on top and mirror at the bottom clamped in a steel frame. On first plane ears bonded on both the masses and two fibres welded.



**Figure 2.** Detail of two fibres welded on tips of a mirror ear.

Q measurements of the greased mass were taken in another facility. In this facility the mass was suspended by a single steel wire. After the modes were been excited by an electrostatic actuator, the displacement of the front face was detected using a Michelson interferometer. More details about this facility can be found in [4]. Due to the short decay time of all the detected modes, Q measurements were firstly taken by measuring the transfer function at frequencies close to the resonances and later the peak widths were extracted. Later, a recording oscilloscope was used to sample the ring down of each detected mode and then the decay time was derived. The transfer function



**Figure 3.** Displacement spectral densities. Curves a) and b) are recorded from the interferometer output before and after putting grease on one of the masses respectively. Curves c) and d) are estimations using a  $Q$  of 1800 for the greased mass and  $1.2 \cdot 10^5$  for the others.

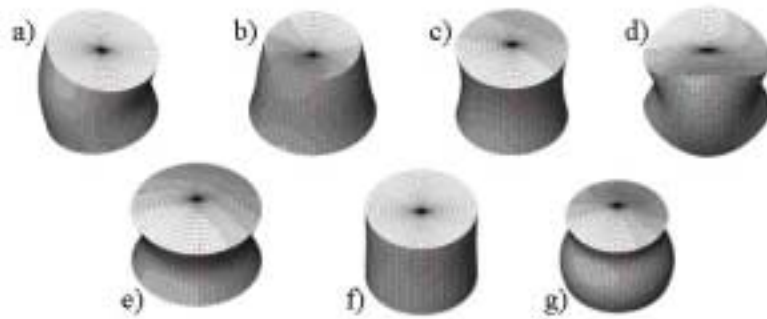
measurement could be affected by the presence of split modes so close each other as not be measured as separate modes. The two set of measurements gave consistent results. After removing the grease,  $Q$  measurements were taken again.

Based on a  $Q$  average value of 1800 for the greased mass and  $1.2 \cdot 10^5$  for the other clean masses, internal thermal noise was estimated using a semi-infinite homogeneous-loss model as in ref. [5]. The results of this calculations are shown in Figure 3. The agreement is good but the theoretical model used is not correct in principle because the grease introduces a non homogeneous loss. As showed by N.Yamamoto [6][7] thermal noise predictions based on homogeneous loss model can differ substantially from those based on an inhomogeneous one. On the other hand, grease is a material having very different mechanical constants from those of silica, and the layer so thick (few millimeters) that it may well be that the inhomogeneous model can not be applied in a straightforward manner. Currently, research is being pursued in this area.

### 3. Coating losses

In a gravitational wave interferometric detector like GEO600 the silica masses are coated in order to obtain the desired reflectivity. Different groups in the scientific community have started to measure or estimate the effect of this coating on the internal damping of the suspended masses [9][10]. Multi layer structure and the kind of materials used give sufficient reasons for a possible high loss characteristic for the coating. At the IGR in Glasgow we measured the loss factor of 7 lowest modes (Figure 4) of a Corning 7980

coated mass (127 mm diameter, 100 mm length). The substrate loss is dominated by



**Figure 4.** Shapes of the detected modes. a) Bending; b) 1st Asymmetric; c) Fundamental; d) Clover; e) Symmetric; f) Expansion; g) 2nd Asymmetric.

the structural damping that is independent of frequency. So, differences between loss factor of different modes has to be considered as a result of the coating effect. Loss properties of the coating are assumed to affect the total loss factor  $\phi_t$  shown by a mode through the following relation:

$$\phi_t = \phi_0 + C_{ff} \cdot \phi_{cf} + C_b \cdot \phi_{cb} \quad (1)$$

where  $\phi_0$  is the substrate or bulk loss angle,  $\phi_{cf}$  and  $\phi_{cb}$  are the coating loss factors associated with the front face and with the barrel. In fact, during the coating process, some coating material spilled off and contaminated the barrel in a unknown way. In order to take into account this uncertainty, two different loss factors are assumed for the coating on the front face and on the barrel.  $C_{ff}$  and  $C_b$  are the scaling factors and they are defined as front face to bulk and barrel to bulk energy ratios respectively. A proof of the equation (1) for a general case is given in ref. [11]. Since the energy ratios scale linearly with the coating thickness, at least for thin coatings, both the  $C$  parameters are calculated per  $\mu\text{m}$  coating thickness; values for each mode are listed on Table 1. Finite element analysis was used for the mode shapes and a C-code was developed to calculate the energies.

When all the data were used in a 3-parameter linear fitting, to solve equation (1), the resulting regression parameter R was rather low. The best fitting was obtained by rejecting the data for the symmetric mode and the result is shown in Figure 5. The agreement between expected values and the experimental points is within the experimental errors. The large discrepancy obtained for the symmetric mode may come from a suspension loss particularly effective for this mode.

Using the loss factors per unit thickness coming from the fitting function and knowing that the effective coating thickness was  $6.3\mu\text{m}$ , the following values for the loss factors are obtained:

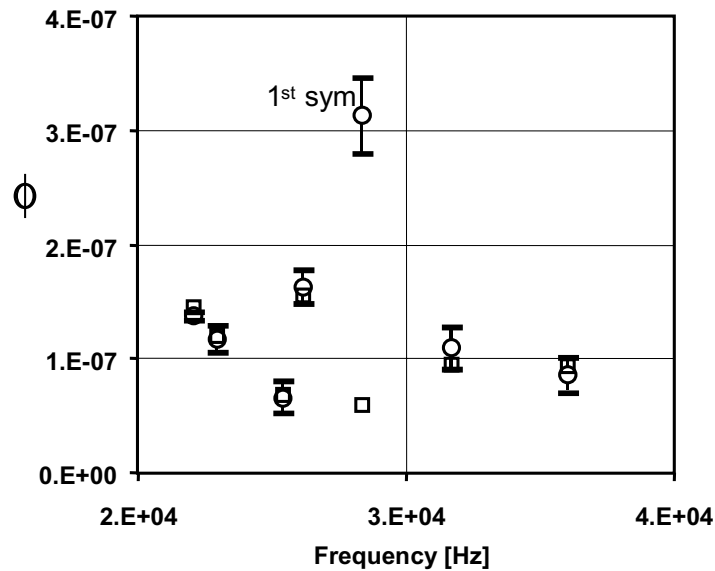
$$\phi_0 = (5.6 \pm 0.9) \cdot 10^{-8}$$

**Table 1.** Summary table showing results obtained for the Corning 7980 mass. Loss factor values are averages of measures taken with different suspensions.  $C_{ff}$  is the front face to bulk energy ratio.  $C_b$  is the barrel to bulk energy ratio. Both are per  $\mu\text{m}$  coating thickness.

Mode	Frequency [Hz]	Loss factor $\times 10^7$	$C_{ff}$ (front face)	$C_b$ (barrel)
Bending	22110	$1.37 \pm 0.04$	$1.67 \cdot 10^{-5}$	$2.00 \cdot 10^{-4}$
1st Asymmetric	22910	$1.17 \pm 0.12$	$1.23 \cdot 10^{-4}$	$3.61 \cdot 10^{-5}$
Fundamental	25380	$0.65 \pm 0.14$	$1.39 \cdot 10^{-7}$	$2.71 \cdot 10^{-5}$
Clover	25180	$1.62 \pm 0.15$	$3.56 \cdot 10^{-5}$	$2.21 \cdot 10^{-4}$
Symmetric	28390	$3.13 \pm 0.33$	$1.05 \cdot 10^{-4}$	$2.12 \cdot 10^{-4}$
Expansion	31710	$1.09 \pm 0.19$	$1.85 \cdot 10^{-5}$	$8.13 \cdot 10^{-5}$
2nd Asymmetric	36050	$0.86 \pm 0.15$	$2.94 \cdot 10^{-6}$	$9.22 \cdot 10^{-5}$

$$\phi_{cf} = (6.3 \pm 1.6) \cdot 10^{-5}$$

$$\phi_{cb} = (6.3 \pm 0.9) \cdot 10^{-5}$$



**Figure 5.** This plot shows the comparison between the measured loss factor for each mode (circle points) and the estimated loss factor (square points) as they come from a 3 parameter linear fitting. The symmetric mode data was taken out from the fitting process because it shows a significant higher loss than the estimated one.

#### 4. Non linear thermoelastic effect

Thermal fluctuations inside a material produce strain fluctuations due to thermal expansion effects (coefficient  $\alpha$ ). This mechanism is responsible for shape fluctuations of mirrors [5] and of suspensions fibres that are seen by the interferometer as displacement noise. Thermal conduction makes this mechanism dissipative and it represents a fundamental process known as thermoelastic damping, studied as first by C. Zener [8].

The mechanical loss angle determined by thermoelastic damping has the following general expression:

$$\phi_Z = \alpha^2 \frac{ET}{C_V} f(\omega) \quad (2)$$

where  $\alpha$  is the linear thermal expansion coefficient,  $E$  the Young's modulus,  $T$  the temperature,  $C_V$  is the heat capacity per unit volume and finally  $f(\omega)$  is the frequency dependent part that is defined by the dimensions of the sample and by heat conductivity  $K$  and capacity  $C_V$  of the material used. The coefficient in front of  $f(\omega)$  in the previous equation gives the strength of the damping. For silica this coefficient is about  $3.6 \cdot 10^{-6}$  which is high compared to the structural loss factor that can be as low as  $1 \cdot 10^{-7}$  in thin fibres. This comparison becomes more relevant if one considers that for silica fibres the thermoelastic pick has its maximum on the range of tens of Hz where the effects of the pendulum thermal noise are still relevant in comparison with the ones coming from internal thermal noise of mirrors.

There is another mechanism to convert temperature fluctuations to strain ( $\epsilon$ ) fluctuations. In this mechanism the stress-strain relations have an explicit temperature dependence through their elastic coefficients. Young's modulus variations contribute to the strain field ( $\sigma$ ) as a second order effect except when a large static stress is present in the material. This is the case for the suspension wires or fibres that are stressed by the weight of a test mass. For longitudinal deformations the stress-strain relation reads  $\epsilon = \sigma/E$ . If a variation  $\delta E \ll E$  of the Young's modulus is considered, then  $\delta\epsilon \simeq -\epsilon(\delta E/E)$ . In the case of a large static strain, from the previous relation, the Young's modulus variation produces a first order effect on the dynamical strain. Moreover the sign of  $\delta\epsilon$  respect to  $\delta E$  depends on the static strain applied.

For a stressed body, then, one can speak about an effective thermal expansion coefficient  $\alpha_e = \alpha - \beta(\sigma_0/E)$  where  $\beta = (dE/dT)/E$  is the linear thermoelastic coefficient. It is possible to demonstrate [12] that the loss factor becomes:

$$\phi_{NLT} = \left( \alpha - \beta \frac{\sigma_0}{E} \right)^2 \frac{ET}{C_V} f(\omega) \quad (3)$$

For silica  $\beta = 2 \cdot 10^{-4}$  1/K,  $\alpha = 5.5 \cdot 10^{-7}$  1/K,  $E = 72$  GPa. Assuming these constants, thermal expansion is completely compensated by Young's modulus variation

with a static stress of only  $\sigma_0 = 200$  MPa. In this condition the overall thermoelastic effect is nulled.

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